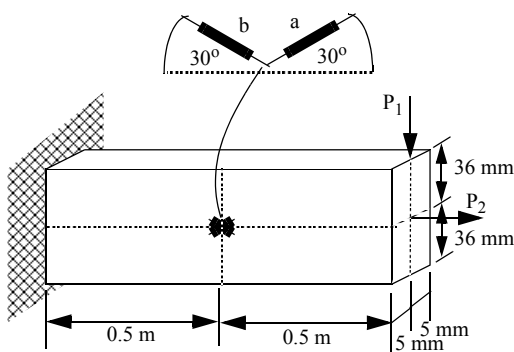


1. The strain gages mounted on the middle surfaces of a cantilever beam recorded the following strain values:  $\epsilon_a = 1333 \mu$   $\epsilon_b = 1592 \mu$ . Determine the loads  $P_1$  and  $P_2$ . The Modulus of Elasticity is 200 GPa and the Poisson's ratio is 0.3.



$P_1 = \dots\dots\dots$

$P_2 = \dots\dots\dots$

2. The stress at a point is given by the stress matrix:

$$\begin{bmatrix} 50 & 0 & 0 \\ 0 & 20 & -20 \\ 0 & -20 & 40 \end{bmatrix} \text{ksi}$$

- (a) Determine the three stress invariants.

$I_1 = \dots\dots\dots$

$I_2 = \dots\dots\dots$

$I_3 = \dots\dots\dots$

- (b) Determine the orientation of principal direction associated with principal stress 7.64 ksi (T).

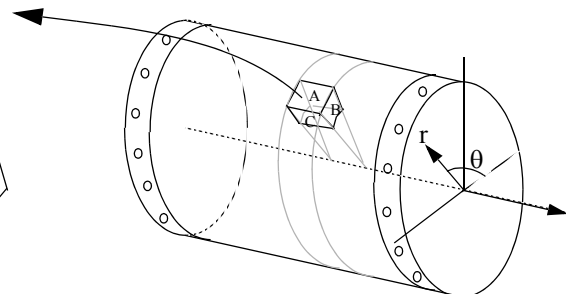
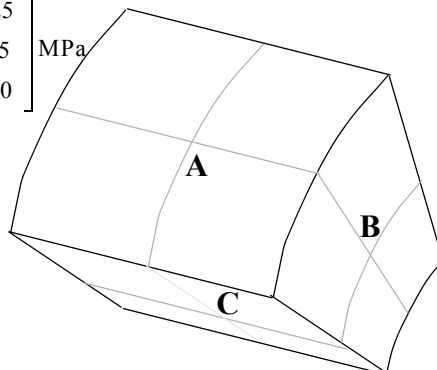
$\theta_x = \dots\dots\dots$   $\theta_y = \dots\dots\dots$   $\theta_z = \dots\dots\dots$

- (c) A crack was discovered on a plane that has an outward normal at angles of  $65^\circ$ ,  $55^\circ$ , and  $45.4^\circ$  with the x, y and z axis respectively. The critical stress intensity factor for the material is  $22 \text{ksi}\sqrt{\text{in}}$ , what would be the critical crack length on that plane.

$2a_{\text{crit}} = \dots\dots\dots$

3. (a) Show the stress components in the r,  $\theta$ , and x cylindrical coordinate system on the A, B, and C faces of the stress element shown.

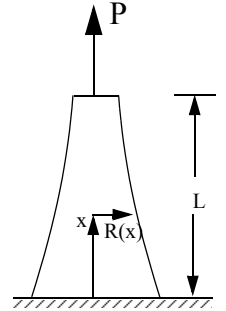
$$\begin{bmatrix} \sigma_{rr} = -145 & \tau_{r\theta} = 100 & \tau_{rx} = -125 \\ \tau_{\theta r} = 100 & \sigma_{\theta\theta} = 160 & \tau_{\theta x} = 165 \\ \tau_{xr} = -125 & \tau_{x\theta} = 165 & \sigma_{xx} = 150 \end{bmatrix} \text{MPa}$$



- (b) A thin-walled cylindrical gas vessel has a mean radius of 3 feet and a wall thickness of 1/2 inch. The yield stress of the material is 30 ksi. Using Von-Mises failure criterion determine the maximum pressure of the gas inside the cylinder if yielding is to be avoided.

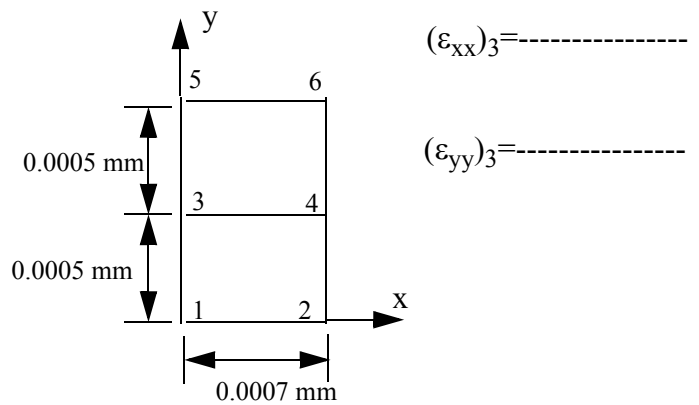
$p_{\text{max}} = \dots\dots\dots$

(c) The radius of a tapered axial rod varies as  $R = \sqrt{2L - x}$ . Determine the extension of the rod due to the axial force  $P$  in terms of the variables  $P$ ,  $L$  and  $E$ . Weight is not a consideration in this problem.

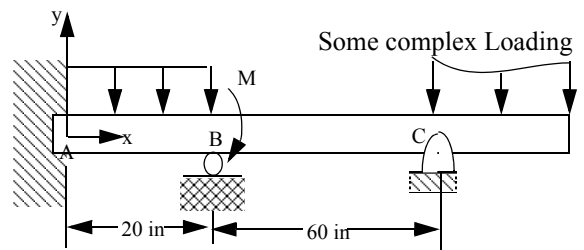


(d) The displacement  $u$  and  $v$  in the  $x$  and  $y$  direction, respectively, are given at six points as shown. Using finite difference approximation find the normal strains  $\epsilon_{xx}$  and  $\epsilon_{yy}$  at point 3. Indicate which finite difference method you used?

Point	$u$ ( $\mu\text{mm}$ )	$v$ ( $\mu\text{mm}$ )
1	0.000	0.000
2	-0.112	0.144
3	0.112	0.176
4	-0.032	0.224
5	0.128	0.384
6	-0.048	0.336



(e) The displacement in the  $y$  direction in section AB is given by  $v_1(x) = -3(x^4 - 20x^3)(10^{-6})$  in and in BC by  $v_2(x) = -8(x^2 - 100x + 1600)(10^{-3})$  in. If the bending rigidity ( $EI$ ) is  $120 (10^6)$  lbs-in<sup>2</sup>. Determine the reaction force at B.



$R_B = \dots\dots\dots$

- 1.  $P_1 = 11 \text{ kN}$                        $P_2 = 312 \text{ kN}$
- 2a.  $I_1 = 110 \text{ ksi}$                        $I_2 = 3400 \text{ ksi}^2$                        $I_3 = 20,000 \text{ ksi}^3$
- 2b.  $\theta_x = 90^\circ$                        $\theta_y = 31.7^\circ$                        $\theta_z = 58.3^\circ$
- 2c.  $2a_{\text{crit}} = 0.422 \text{ inch}$
- 3b.  $p_{\text{max}} = 480 \text{ psi}$
- 3c.  $0.221P/(\pi E)$
- 3d.  $(\epsilon_{xx})_3 = -205.7 \mu$  Forward                       $(\epsilon_{yyx})_3 = 384 \mu$  Central
- 3e.  $R_B = 129.6 \text{ kips}$